

**Mechanical Systems Laboratory: Lecture 8**  
**Brief Review of Stability; PD Position Control of a Robot Arm**

**1. Brief Review of Stability**

Stability refers to the concept of whether a system's performance "blows up" or converges to some value. What are some applications in which stability analysis is very important?

Three types of stability are:

The location of the poles of the transfer function determine the type of stability. Why? Consider a second-order system:

Remember, the inverse Laplace transform of the transfer function is the impulse response:

Case 1: $\text{Re}\{p_i\} < 0$		
Case 2: $\text{Re}\{p_i\} = 0$		
Case 3: $\text{Re}\{p_i\} > 0$		

So the location of the poles in the complex plane determines the type of response of the system.

*Exercise 1:* Label the complex plane with the following words:

stable, unstable, marginally stable, oscillation, no oscillation, faster, slower, higher frequency oscillation, lower frequency oscillation

**2. PD Position Control of a Robot Arm (P = proportional, D = Derivative)**

Position control – most common industrial control system

Can you think of some applications?

Consider a one-joint robot arm:

Assume: 1) no friction or gravity; 2) we have a controller that can apply any torque that we want; 3) we can sense  $\theta$  (for example, with a potentiometer)

*Exercise 2:* Design a proportional feedback controller to position the robot arm at  $\theta = \theta_d$ , find its transfer function, and analyze its stability

*Exercise 3:* Design a way to fix the problem. What kind of hardware would you need?

Two approaches to sensing angular velocity:

- 1)
- 2)

What are the dynamics and transfer function of the robot with the new controller?

How are the gains  $K_p$  and  $K_v$  related to the natural frequency and damping ratio?

What is the step response of the system?

$$\zeta = 0 \quad \theta(t) = 1 - \cos(\omega_n t)$$

$$0 < \zeta < 1 \quad \theta(t) = 1 - \frac{e^{-\zeta\omega_n t}}{\sqrt{1-\zeta^2}} \sin(\omega_d t + \tan^{-1} \frac{\sqrt{1-\zeta^2}}{\zeta}) \quad \omega_d = \omega_n \sqrt{1-\zeta^2}$$

$$\zeta = 1 \quad \theta(t) = 1 - e^{-\omega_n t} (1 + \omega_n t)$$

$$\zeta > 1 \quad \text{sum of two exponentials – see book; can neglect one exponential if } \zeta > 2$$

Note: the damping ratio determines whether the system oscillates (i.e. whether the poles have an imaginary part)

*Exercise 4:* Plot what the step response of the system would look like for different values of the damping ratio and natural frequency.

Notes: Overdamped systems are “sluggish”

Among systems responding without overshoot, critically damped systems exhibit the fastest response.

Underdamped systems with  $0.5 < \zeta < .8$  get close to final value more rapidly than critically damped or overdamped systems.

The settling time of an underdamped (or critically damped) system is:

*Exercise 5:* Given a one-joint robot arm (no friction, no gravity) with  $J = 1 \text{ kgm}^2$ . Design a PD position controller such that the robot finishes 95% of a commanded step-function movement in .5 seconds, with no overshoot.